MEMS Packaging Techniques for Silicon Optical Benches

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Why MEMS for optoelectronic packaging?

- high precision required: ±0.5 micron, ±0.7°
- towards parallel assembly
- avoid expense of nanomanipulator





Introduction Why rapid prototyping?

- many new materials
- mechanical design hard

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 - Laser beam 3ns pulses @ 50Hz 0.6mJ/pulse 355nm or 532nm 50pm-diameter spot



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Previous MEMS packaging work: passive



Journal Micromechanics Microengineering 8, 343-360 (1998)





Previous MEMS packaging work: out of plane

• polysilicon multi-layer processes (e.g. SUMMiT, Sandia)



Previous MEMS packaging work: out of plane

- polysilicon multi-layer processes (e.g. SUMMiT, Sandia)
- surface tension: self-assembly



Imperial College http://www.ee.ic.ac.uk/optical/Microsystems.html





3ns-pulse 3.5eV (UV), 50Hz, 10µms⁻¹

0.1mm

3ns-pulse 2.3eV (Green), 50Hz, 10µms⁻¹

0.1mm

3ps-pulse 1.2eV (IR), 50kHz, 10mms Lumera Laser

0.1mm



SiN (2μm) SiN (0.2μm) ta-C(0.1μm) Bare silicon threshold

















- resonance frequency measurement¹
- electrostatic pull-in²
- microbeam deflection with nanoindenter³

M Madou: <u>Fundamentals of Microfabrication</u>, p270
 Journal MEMS, **6** 2 1997 pp107–118
 WD Nix: *Measurement of Mechanical Properties in Small Dimensions by Microbeam Deflection*, Stanford

- resonance frequency measurement¹
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- microbeam deflection with nanoindenter³
- scanning profilometer along microbeam





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 $Z = F\{(x-x_0-L_u)^3/3EI + (x-x_0-L_u)^2L_u/EI_u + (x-x_0-L_u)L_u^2/EI_u + L_u^3/3EI_u)\}$ = $Fx^3/3EI + O(x^2)$



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Extracting Young's Modulus of thin films — additional problems

- anticlastic curvature affects effective stiffness
- stylus force varies with deflection
- local indentation
- beam twisting

Extracting maximum stress of thin films









Problem abstraction

| | | | θ, x, y: in plane z, φ, ψ: out of | | | | |
|------------|----|-------------------|--------------------------------------|--------------------------------------|------------------------|---|--|
| | | 0 | 1 | 2+ | 3 | plane | |
| Linear DoF | 0 | filter | | | | * x or y ⁺ includes θ plus φ or ψ [§] translation in x or y, plus z [#] positioning in x and y | |
| | 1* | | | plane mirror | diffraction grating | | |
| | 2 | | | detector§ | triangular prism# | | |
| | 3 | spherical lens | | parabolic lens or mirror; emitter | | | |

Specification

- Alignment precision: 0.1dB coupling loss requires 0.5µm and 0.7° but throw must be larger
- 2. Components to be held sub-millimetre: lenses, mirrors and fibres
- 3. Slop: component dimension tolerances up to 10%
- 4. Up to 2 linear and 3 angular degrees of freedom or vice versa
- 5. Power and area budgets: debatable



DRIE structure Lens manipulator



DRIE structure Lens manipulator



Out-of-plane bistable clamps





Proposed microclips

When a component is inserted, the microcantilevers deflect and hold the component in static equilibrium.



Fabrication

- 1. deposit thin film on to wafer
- 2. photolithography or laser micromachining to define cantilevers
- 3. anisotropic etch through wafer

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Deflection of Microbeams

- A thin beam subject to a tip displacement constraint assumes a shape that minimises energy.
- To obtain a minimum energy state we must vary the shape function of the cantilever until the integral energy function reaches a stationary point.



$$E = k \int_{0}^{L} F(y, \frac{dy}{dx}, x) dx$$
$$E_{tot} = \int M \kappa ds$$

Discretisation

- The integral expression can be approximated by a series expression.
- The system effectively becomes a series of rigid members joined by torsional springs whose spring constant gives the same bending rigidity per unit length as the continuous bar.



Deflection of Microbeams

- Excel contains an optimisation plug-in called Solver. This allows us to
 - Minimise an objective function
 - By changing a set of variables
 - Subject to constraints





Stored Energy

Bar Angles $(\theta_0 ... \theta_n)$

Tip Displacement + segment angle

| Node | θ degrees | heta radians | Un | Vn | $\Delta \theta_n$ | Energy |
|------|------------------|--------------|------|-------|-------------------|-----------|
| 0 | 0.00 | 0 | 0.00 | 0.00 | | |
| 1 | 1.32 | 0.0230469 | 0.03 | 0.00 | 0.0230469 | 4.46E-03 |
| 2 | 2.64 | 0.0460318 | 0.07 | 0.00 | 0.0229848 | 4.44E-03 |
| 3 | 3.95 | 0.0688915 | 0.10 | 0.00 | 0.0228598 | 4.39E-03 |
| 4 | 5.25 | 0.0915638 | 0.13 | -0.01 | 0.0226723 | 4.32E-03 |
| 5 | 6.53 | 0.1139874 | 0.17 | -0.01 | 0.0224236 | 4.22E-03 |
| 6 | 7.80 | 0.1361017 | 0.20 | -0.02 | 0.0221143 | 4.11E-03 |
| 7 | 9.04 | 0.1578469 | 0.23 | -0.02 | 0.0217453 | 3.97E-03 |
| 8 | 10.27 | 0.1791647 | 0.26 | -0.03 | 0.0213178 | 3.82E-03 |
| 9 | 11.46 | 0.1999985 | 0.30 | -0.03 | 0.0208338 | 3.65E-03 |
| 10 | 12.62 | 0.2202919 | 0.33 | -0.04 | 0.0202934 | 3.46E-03 |
| 11 | 13.75 | 0.2399913 | 0.36 | -0.05 | 0.0196994 | 3.26E-03 |
| 12 | 14.84 | 0.2590443 | 0.39 | -0.06 | 0.0190529 | 3.05E-03 |
| 13 | 15.89 | 0.2774013 | 0.43 | -0.07 | 0.018357 | 2.83E-03 |
| 14 | 16.90 | 0.2950133 | 0.46 | -0.08 | 0.017612 | 2.61E-03 |
| 15 | 17.87 | 0.3118345 | 0.49 | -0.09 | 0.0168212 | 2.38E-03 |
| 16 | 18.78 | 0.3278219 | 0.52 | -0.10 | 0.0159874 | 2.15E-03 |
| 17 | 19.65 | 0.342934 | 0.55 | -0.11 | 0.0151121 | 1.92E-03 |
| 18 | 20.46 | 0.3571318 | 0.58 | -0.12 | 0.0141977 | 1.69E-03 |
| 19 | 21.22 | 0.370379 | 0.62 | -0.13 | 0.0132472 | 1.47E-03 |
| 20 | 21.92 | 0.3826425 | 0.65 | -0.14 | 0.0122636 | 1.26E-03 |
| | | | | | _ | |
| | | | | | Σ Energy | 6.345E-02 |

IEEE 2002 Electronics Components and Technology Conference, San Diego, U.S.A., s19p3, pp. 1-7 (2002)

Processing variations and probabilistic design

- For optical benches components must be precisely aligned in the vertical plane.
- Process variability can lead to component misalignment.



Processing variations and probabilistic design



Towards 'active' clips



Thicknesses a_1 , a_2 Thermal expansivities are α_1 , α_2 Moduli E_1 , E_2 Increase in temperature ΔT

$$E_1/E_2 = n$$

$$a_1 + a_2 = t$$

$$a_1/a_2 = m$$

$$K = 6(1 + m)^2 / [3(1 + m)^2 + (1 + mn)(m^2 + 1/mn)]$$

Thermal curvature is given by:

 $\kappa = K(\alpha_2 - \alpha_1) \Delta T/t$





No misalignments

2 0.0002 0.0002 0.0002 0.0008 0.0004 0.0006 0.0008 0.0001 0.00012 0.00014

Local energy minimum with antisymmetric clip orientations

rotation $\psi = -11.5^{\circ}$





20 µm front-to-back misalignment…

rotation $\psi = -2.1^{\circ}$ x ≈ 0



No misalignments

20 µm front-to-back misalignment…

rotation $\psi = -2.1^{\circ}$ x ≈ 0 ... countered by 20K temp increase in red clip. (150nm Ni/SiN_x)

rotation $\psi = 0.4^{\circ}$ x ≈ 0



rotation $\psi \sim 2^{\circ}$ x ~ -5 μ m



Why consider inflatable MEMS?

- sharp edges unstressed
- films in tension mean larger holding forces
- inherent damping
- fluid could solidify

Explore the possibilities of making inflatable microballoons from elastic films, either welded together or spun on to a rigid substrate.





Conclusions

Laser micromachining

- cutting slow / refinement fast
- material damage can probably be controlled
- UV/silicon nitride combination ideal

Modulus extraction

- target accuracy better than 20%
- noise in data remains largest problem

Deep reactive ion etching

+ strong + well-characterised – large footprint

Thin-film microclips

+ simple + compact - max. stress high

Inflatable MEMS

- + manipulate and fix in one step
- + strength from inflation not high modulus unproven